Relaxed CBF-based control for inspection with underactuated USVs

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Introduction



Motivation: Survey, monitoring, or interception applications require:

- approaching target as closely as possible,
- while guaranteeing a minimum safe standoff distance,
- global tracking stability with minimal computational effort,
- and dynamically feasible control inputs for underactuated surface vessels.

Approach: Use of Control Barrier Functions

Problem Formulation

$$\begin{split} \dot{\boldsymbol{\eta}} &= \boldsymbol{J}(\boldsymbol{\eta})\boldsymbol{v}, \\ \boldsymbol{M}\dot{\boldsymbol{v}} &= -\boldsymbol{C}(\boldsymbol{v})\boldsymbol{v} - \boldsymbol{D}(\boldsymbol{v})\boldsymbol{v} + \boldsymbol{\tau}. \\ \\ \boldsymbol{u}\cos\psi - \boldsymbol{v}\sin\psi \\ \boldsymbol{u}\sin\psi + \boldsymbol{v}\cos\psi \\ \boldsymbol{r} \\ \boldsymbol{f}_{\boldsymbol{\chi}} \\ \boldsymbol{f}_{\boldsymbol{y}} \\ \boldsymbol{f}_{\boldsymbol{\psi}} \\ \end{split} \right] + \begin{bmatrix} 0 & 0 \\ 0 & 0 \\ 0 & 0 \\ \frac{1}{m_{11}} & 0 \\ 0 & \frac{a_y}{m_{33}} \\ 0 & \frac{a_\psi}{m_{33}} \\ \end{bmatrix} \boldsymbol{\tau} \\ \boldsymbol{\tau} = \begin{bmatrix} \tau_{\boldsymbol{\chi}} & \tau_{\boldsymbol{\psi}} \end{bmatrix}^{\boldsymbol{T}}. \end{split}$$



Control Objective: Stable trajectory tracking and target approach for underactuated USVs.

Candidate Lyapunov Functions

Consider the nonlinear system

 $\dot{\boldsymbol{x}} = \boldsymbol{f}(\boldsymbol{x}) + \boldsymbol{G}(\boldsymbol{x})\boldsymbol{u},$

(1)

which is affine in the control input. Lyapunov Stability Theorem Suppose there is a function

$$V(\mathbf{x}) > 0, \quad \forall \mathbf{x} \in D, \mathbf{x} \neq 0,$$

where

$$\dot{V}(\boldsymbol{x}) < 0, \quad \forall \boldsymbol{x} \in D, \boldsymbol{x} \neq 0,$$

then (1) is asymptotically stable.

Set of points in V shrinks in time since $\dot{V} < 0$.



Standard Control Barrier Functions

Safe Set (convex)

$$\begin{array}{rcl} \mathcal{C} &=& \{ \pmb{x} \in D \subset \mathbb{R}^n : B(\pmb{x}) \geq 0 \}, \\ \\ \partial \mathcal{C} &=& \{ \pmb{x} \in D \subset \mathbb{R}^n : B(\pmb{x}) = 0 \}, \\ \\ \\ \mathrm{Int}(\mathcal{C}) &=& \{ \pmb{x} \in D \subset \mathbb{R}^n : B(\pmb{x}) > 0 \}, \end{array}$$

Control Barrier Function

B is a CBF if

$$\sup_{\boldsymbol{u}\in\mathcal{U}}\left[\frac{\partial B}{\partial \boldsymbol{x}}\boldsymbol{f}+\frac{\partial B}{\partial \boldsymbol{x}}\boldsymbol{G}\boldsymbol{u}\right]\geq-\alpha(B(\boldsymbol{x}))\quad(2)$$

Use of $\alpha(\cdot)$ (a \mathcal{K}_{∞} function) keeps safe set from shrinking.



Theorem: If B(x) is a CBF, any locally Lipschitz continuous u satisfying (3) renders C safe for (1).

Relaxed Control Barrier Functions

Consider system (1) and safe set C. A class C^1 function $B: D \times D \to \mathbb{R}$, where $D \subset \mathbb{R}^n$, is a relaxed control barrier function if:

- 1 $B(\tilde{x}, \tilde{x}_0) \ge 0$ for all $\tilde{x}_0 \in C$.
- 2 *B* is proper, i.e. $\{\tilde{\mathbf{x}}, \tilde{\mathbf{x}}_0 \mid B(\tilde{\mathbf{x}}, \tilde{\mathbf{x}}_0) \leq L\}$ is compact for any $L \geq 0$.
- **3** For any continuous $\boldsymbol{u}_{c} \in \mathbb{R}^{m}$, there exist non-negative constants $\alpha, \beta \geq 0$, so that

$$\inf_{\boldsymbol{u}_{c}\in\mathbb{R}^{m}}\dot{B} = \inf_{\boldsymbol{u}_{c}\in\mathbb{R}^{m}}\left[\frac{\partial B}{\partial\tilde{\boldsymbol{x}}}\cdot\dot{\tilde{\boldsymbol{x}}}+\frac{\partial B}{\partial\tilde{\boldsymbol{x}}_{0}}\cdot\dot{\tilde{\boldsymbol{x}}}_{0}\right], \\ < \alpha B(\tilde{\boldsymbol{x}},\tilde{\boldsymbol{x}}_{0})+\beta.$$
(3)

The trajectory tracking safety-critical control input is designed by solving the optimization problem

$$\min_{\boldsymbol{u}_c} \|\boldsymbol{u}_c - \boldsymbol{u}_t\|^2, \qquad (4a)$$

s.t.
$$\dot{B} - \alpha B(\tilde{\mathbf{x}}, \tilde{\mathbf{x}}_0) - \beta < 0.$$
 (4b)

Trajectory Tracking Safety-Critical Controller

Consider system (1). Let u_t be a trajectory tracking control input,

$$I := \left(\frac{\partial B}{\partial \tilde{\mathbf{x}}} + \frac{\partial B}{\partial \tilde{\mathbf{x}}_0}\right) \cdot \left[\mathbf{f}(\mathbf{x}) + \mathbf{g}(\mathbf{x})\mathbf{u}_t\right] - \frac{\partial B}{\partial \tilde{\mathbf{x}}} \cdot \dot{\mathbf{x}}_d - \frac{\partial B}{\partial \tilde{\mathbf{x}}_0} \cdot \dot{\mathbf{x}}_0,$$
(5)
$$L_g B := \left(\frac{\partial B}{\partial \tilde{\mathbf{x}}} + \frac{\partial B}{\partial \tilde{\mathbf{x}}_0}\right) \cdot \mathbf{g}(\mathbf{x}),$$
(6)

and

$$J := \alpha B(\tilde{\mathbf{x}}, \tilde{\mathbf{x}}_0) + \beta.$$
(7)

The control input

$$\boldsymbol{u}_{c} = \begin{cases} \boldsymbol{u}_{t}, & l \leq J, \\ \boldsymbol{u}_{t} - (l - J) \frac{L_{g} B^{T}}{\|L_{g} B\|^{2}}, & l > J, \end{cases}$$
(8)

is the solution of the optimal control Problem (4).

Control Design: Underactuated USV

Tracking errors: $\tilde{x} = x - x_d$ and $\tilde{y} = y - y_d$

Distance to target: $\tilde{x}_0 := x - x_0$ and $\tilde{y}_0 := y - y_0$

Approach:

- underactuated USV can apply both a surge force and yaw moment, but cannot directly apply a force in the sway direction
- 2 motion in the sway and yaw directions coupled via the added mass terms
- **3** use coupling to find yaw moment τ_{ψ} required for virtual control input \dot{r}_c that relates yaw acceleration to the sway acceleration, such that commanded control inputs dynamically feasible.
- 4 control design includes two backstepping stages + Dynamic Surface Control
 - design kinematic position tracking controller \rightarrow virtual control inputs
 - design physical control inputs taking nonholonomic second order (acceleration) constraints into account
 - use DSC to compute time derivatives of virtual control inputs

Control Design: Kinematic Controller

Take virtual control inputs to be

$$\dot{x}_t = -k_x \tilde{x} + \dot{x}_d \tag{9}$$

and

$$\dot{y}_t = -k_y \tilde{y} + \dot{y}_d, \tag{10}$$

where $k_x > 0$ and $k_y > 0$ are constants. Trajectory tracking control input (virtual) is given by $\boldsymbol{u}_t = \begin{bmatrix} u_t & v_t \end{bmatrix}^T$, where

$$u_t = \dot{x}_t \cos \psi + \dot{y}_t \sin \psi,$$

$$v_t = -\dot{x}_t \sin \psi + \dot{y}_t \cos \psi.$$
(11)

Control Design: Safety-Critical Kinematic Controller

Use modified form of relaxed CBF proposed by Igarashi & Nakamura (2018), take

$$B(\tilde{\boldsymbol{x}}, \tilde{\boldsymbol{x}}_0) = \frac{1}{2} \left\{ \frac{1}{\left[\|\tilde{\boldsymbol{x}}_0\|^{1/k} - \|\tilde{\boldsymbol{x}}_0\|_{\min}^{1/k} \right]} + \tilde{\boldsymbol{x}}^T \tilde{\boldsymbol{x}} \right\},$$
(12)

where k selected to modify agressiveness of safety critical controller as the USV approaches safety "barrier" located at $\|\tilde{x}_0\| = \|\tilde{x}_0\|_{\min}$.



Kinematic equation of motion for the virtual control inputs u_c and v_c is

$$\begin{bmatrix} \dot{x}_c \\ \dot{y}_c \end{bmatrix} = \begin{bmatrix} \cos\psi & -\sin\psi \\ \sin\psi & \cos\psi \end{bmatrix} \begin{bmatrix} u_c \\ v_c \end{bmatrix}.$$
 (13)

Virtual tracking control inputs are in the control affine form of (1) with f(x) = 0 and $G = g(\psi)$, where

$$\boldsymbol{g}(\psi) = \begin{bmatrix} \cos\psi & -\sin\psi \\ \sin\psi & \cos\psi \end{bmatrix}.$$
 (14)

Control Design: Dynamic safety-critical tracking control

To determine physical control inputs τ_x and τ_{ψ} that generate virtual control input \boldsymbol{u}_c , define velocity error surfaces $\tilde{\boldsymbol{u}} := \boldsymbol{u} - \boldsymbol{u}_c$, $\tilde{\boldsymbol{v}} := \boldsymbol{v} - \boldsymbol{v}_c$ and $\tilde{\boldsymbol{r}} := \boldsymbol{r} - \boldsymbol{r}_c$. Take virtual control input $\dot{\boldsymbol{r}}_c$ and physical control inputs τ_x and τ_{ψ} to be

$$\dot{r}_{c} = \frac{a_{\psi}}{a_{y}} \left[-k_{v} \tilde{v} - f_{y} + \dot{v}_{c} \right] + f_{\psi},$$

$$\tau_{x} = m_{11} \left[-k_{u} \tilde{u} - f_{x} + \dot{u}_{c} \right],$$

$$\tau_{\psi} = \frac{m_{33}}{a_{\psi}} \left[-k_{r} \tilde{r} - f_{\psi} + \dot{r}_{c} \right],$$
(15)

where $k_u > 0$, $k_v > 0$ and $k_{\psi} > 0$ are constants.

Control Design: Dynamic Surface control

 \hat{u}_c and \hat{v}_c filtered estimates of u_c and v_c Let $\tilde{u} := u - \hat{u}_c$, $\tilde{v} := v - \hat{v}_c$, $\tilde{u}_c := \hat{u}_c - u_c$ and $\tilde{v}_c := \hat{v}_c - v_c$

Use exact expression for $\dot{r}_c \rightarrow \tilde{r} := r - r_c$, as before.

Now, take the virtual control input \dot{r}_c and physical control inputs τ_x and τ_{ψ} to be

$$\dot{r}_{c} = \frac{a_{\psi}}{a_{y}} \left[-k_{v} \left(v - v_{c} \right) - f_{y} + \dot{\hat{v}}_{c} \right] + f_{\psi}$$

$$\tau_{x} = m_{11} \left[-k_{u} \left(u - u_{c} \right) - f_{x} + \dot{\hat{u}}_{c} \right],$$

$$\tau_{\psi} = \frac{m_{33}}{2} \left[-k_{r} \left(r - r_{c} \right) - f_{\psi} + \dot{r}_{c} \right].$$

,

(16)

$$= \frac{m_{33}}{a_{\psi}} \left[-k_r \left(r-r_c\right)-f_{\psi}+\dot{r}_c\right].$$

Closed loop error system for the dynamics of the system is

$$\dot{\tilde{u}} = -k_{u}\tilde{u} - k_{u}\tilde{u}_{c},$$

$$T_{d}\dot{\tilde{u}}_{c} = -\tilde{u}_{c},$$

$$\dot{\tilde{v}} = -k_{v}\tilde{v} - k_{v}\tilde{v}_{c},$$

$$T_{d}\dot{\tilde{v}}_{c} = -\tilde{v}_{c},$$

$$\dot{\tilde{r}} = -k_{r}\tilde{r},$$

$$(17)$$

where $T_d \in (0, 1)$ is filter time coefficient of the filter and $\hat{u}_c(0) = 0$ and $\hat{v}_{c}(0) = 0$.

Representative Simulations

- Desired trajectory straight-line constant speed 1.4 m/s
- Control parameters manually tuned

Parameter	Value
α	0.75
eta	1
k	5
k_x, k_y	0.5
k_u, k_v, k_r	0.075
а	0.5
$\ \mathbf{x}_0\ _{\min}$	5.0
T_d	0.01



Representative Simulations

- Desired trajectory straight-line constant speed 1.4 m/s
- (Same) manually tuned control parameters

Parameter	Value
α	0.75
eta	1
k	5
k_x, k_y	0.5
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T_d	0.01



Representative Simulations



Comparison with Nonlinear MPC



Concluding Remarks

- Proposed trajectory tracking safety-critical controller for closest safe approach by an underactuated USV with nonholonomic dynamic (acceleration) motion constraints
- analytical solution to the optimization problem, instead of online optimization computationally lightweight
- modification of relaxed control barrier function of Igarashi & Nakamura (2018), permits the safety critical control to start acting sooner and more gradually
- not as smooth/precise as MPC, but much simpler to configure and less computationally intense
- Future work:
 - Backstepping leads to a nonlinear PD controller opens possibility of robust approaches
 - reduce actuator saturation when the safety-critical controller is deactivated
 - extend the proposed approach to handle model uncertainty, exogenous disturbances and multiple targets